

Fig. 5 Turbulence profiles at the trailing edge.

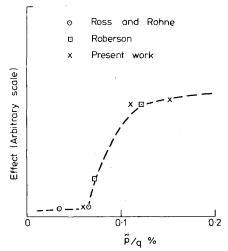


Fig. 6 A model for the effect of pressure fluctuations.

The boundary-layer velocity profiles at the trailing edge of the model for the three levels of pressure fluctuations are shown in Fig. 4. There is a considerable change in the velocity profiles when the levels of \widetilde{p}/q were changed from .006 to .011. The changes, with further increases in \widetilde{p}/q , are not noticeable.

Figure 5 shows the turbulence profiles at the trailing edge for the three levels of \widetilde{p}/q . A typical turbulence profile obtained in an attached turbulent boundary layer at zero pressure gradient is also shown for comparison. The levels of turbulence in the separated layer are generally an order higher than those in an attached boundary layer. The levels are sensitive to the changes in \widetilde{p}/q . With the increase in pressure fluctuation levels, the turbulence initially increases and then decreases. However, the changes in the freestream turbulence levels are insignificant for all levels of \widetilde{p}/q remaining constant at about 2.4%

The general effect of pressure fluctuations on the shock/boundary-layer interaction is shown schematically in Fig. 6. It is suggested that there is virtually no effect of pressure fluctuations up to a level of about 0.6%. With the increase in pressure fluctuations, whether the effect beyond this level is gradual or sudden is uncertain and needs further investigation.

Acknowledgment

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Radiation Shape Factors between End Plane and Outer Wall of Concentric Tubular Enclosures

C. P. Minning*
Hughes Aircraft Company, Culver City, Calif.

Nomenclature

x,y,z	= Cartesian coordinates of dA_1 , cm		
l_1, m_1, n_1	= cosines (i.e., direction cosines) of the angles		
2. 2. 1	between the normal to dA_I and the x, y, and z		
	axes, respectively		
x_2, y_2, z_2	= Cartesian coordinates of a point on the		
	periphery of surface 2, cm		
S	= distance between dA_I and a point on the		
	periphery of surface 2, cm		
r	= radial coordinate in plane of dA_1 , cm		
r_o	= radius of outer tube, cm		
$\frac{r_i}{\theta}$	= radius of inner tube, cm		
$\dot{\theta}$	= angular displacement from x axis, rad		
ω	= view angle, rad		
h	= height of concentric tubes, cm		

Introduction

N radiation heat-transfer calculations for diverse applications such as gas turbine combustion chambers, rotary-kiln dryers, and spin-stabilized spacecraft, it is often desired to calculate the shape factor between an annular-disk end plane and the outer wall of an enclosure formed by concentric cylinders, such as that shown in Fig. 1. If the inner and outer radii of the disk are the same as the radii of the inner and outer cylinders, the shape factor can be calculated by means of the closed-form expressions given in Ref. 2. However, there is no closed-form expression available in the literature for the case where the radii of the annular disk are not equal to the radii of the concentric cylinders.

The contour integral method is used in the present analysis to derive a closed-form expression for the shape factor, $F_{\rm dA_I-A_2}$, from a differential area on the annular-disk end plane to the outer wall of the enclosure. This expression can then be integrated numerically to obtain the desired shape factor from a finite-sized annular disk to the outer wall. Typical results are shown for both configurations.

Analysis

Consider the geometry shown in Figs. 1-3 which illustrate the nomenclature used in this presentation. Throughout this

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^{*}Senior Staff Engineer, Electro-Optical and Data Systems Group.

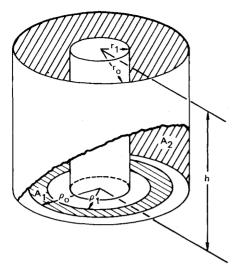


Fig. 1 Nomenclature for shape factors between a finite-sized annular disk and the outer wall of a concentric cylindrical enclosure.

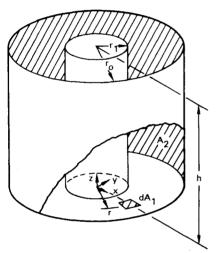


Fig. 2 Nomenclature for shape factor from a differential area to the outer wall of a concentric cylindrical enclosure.

discussion, surface 2 is always considered to be of finite size; surface 1 can be either a differential area or a finite-sized area depending on the problem under consideration.

Sparrow³ has shown that $F_{dA_1-A_2}$ can be expressed as the sum of three contour integrals in the following manner:

$$F_{dA_1 - A_2} = l_1 \oint_C \frac{(z_2 - z) \, dy_2 - (y_2 - y) \, dz_2}{2\pi S^2} + m_1 \oint_C \frac{(x_2 - x) \, dz_2 - (z_2 - z) \, dx_2}{2\pi S^2} + n_1 \oint_C \frac{(y_2 - y) \, dx_2 - (x_2 - x) \, dy_2}{2\pi S^2}$$
(1)

where

$$S^{2} = (x_{2} - x)^{2} + (y_{2} - y)^{2} + (z_{2} - z)^{2}$$
 (2)

Here, C designates integration around the periphery of surface 2. Since the normal to dA_i is parallel to the z axis, $l_i = m_i = 0$, and $n_i = 1$. Hence, only the last term in Eq. (1) is nonzero.

Integration around the periphery of surface 2 is performed in the following sequence: a) along boundary 1, where $\theta = -\omega$

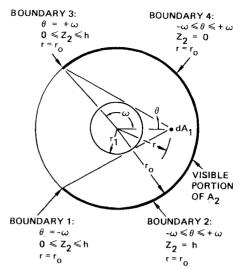


Fig. 3 Visible portion of outer wall of a concentric cylindrical enclosure as seen from the differential area, dA_I .

and z_2 is increased from 0 to h; b) along boundary 2, where $z_2 = h$ and θ is increased from $-\omega$ to $+\omega$; c) along boundary 3, where $\theta = +\omega$ and z_2 is decreased from h to 0; and d) along boundary 4, where $z_2 = 0$ and θ is decreased from $+\omega$ to $-\omega$. Now both dx_2 and dy_2 are equal to zero along boundaries 1 and 3; therefore, the only nonzero integrals in Eq. (1) correspond to boundaries 2 and 4. For both boundaries 2 and 4, $x_2 = r_o \cos\theta$, $dx_2 = -r_o \sin\theta d\theta$, $y_2 = r_o \sin\theta$, $dy_2 = r_o \cos\theta d\theta$, x = r, y = 0, and z = 0. For boundary 2, $z_2 = h$, and for boundary 4, $z_2 = 0$. Only that portion of A_2 which is located in the region $-\omega \leq \theta \leq \omega$ is visible at dA_1 . The view angle ω is defined below (see Fig. 3).

$$\omega = \cos^{-1}(r_1/r) + \cos^{-1}(r_1/r_0)$$
 (3)

Substitution of the appropriate expressions for x,y,z,x_2,y_2 , and z_2 into Eqs. (1) and (2) results in the following expressions:

$$F_{dA_{I}-A_{2}} = \frac{-r_{o}}{2\pi} \int_{-\omega}^{\omega} \frac{(r_{o} - r\cos\theta) d\theta}{r_{o}^{2} - 2rr_{o}\cos\theta + r^{2} + h^{2}}$$
$$-\frac{r_{o}}{2\pi} \int_{\omega}^{-\omega} \frac{(r_{o} - r\cos\theta) d\theta}{r_{o}^{2} - 2rr_{o}\cos\theta + r^{2}}$$
(4)

The integrals in this expression are available in standard references. The desired expression for $F_{\mathrm{d}A_{1}-A_{2}}$ is then found to be

$$F_{dA_{I}-A_{2}} = \frac{I}{\pi} \tan^{-1} \left[\frac{r_{o} + r}{r_{o} - r} \tan(\omega/2) \right]$$

$$- \frac{(r_{o}^{2} - r^{2} - h^{2})}{\pi \sqrt{(r_{o}^{2} + r^{2} + h^{2})^{2} - (4r^{2}r_{o}^{2})}}$$

$$\times \tan^{-1} \left[\frac{\sqrt{(r_{o}^{2} + r^{2} + h^{2})^{2} - 4(r^{2}r_{o}^{2})}}{r_{o}^{2} + r^{2} + h^{2} - 2rr_{o}} \tan(\omega/2) \right]$$
 (5)

Now consider the special case where $r_1 = 0$. For this situation, $\omega = \pi$ and Eq. (5) reduces to

$$F_{\mathrm{d}A_{1}-A_{2}} = \frac{1}{2} \left[1 - \frac{r_{o}^{2} - r^{2} - h^{2}}{\sqrt{(r_{o}^{2} + r^{2} + h^{2})^{2} - 4(r^{2}r_{o}^{2})}} \right]$$
 (6)

which is the same answer obtained when the familiar expression for the shape factor from an off-center differential

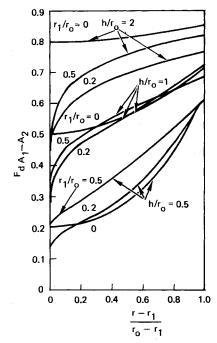


Fig. 4 Shape factor, F_{d,A_I-A_I} , from a differential area to the outer wall of a concentric cylindrical enclosure as a function of radial distance parameter, $(r-r_I)/(r_o-r_I)$.

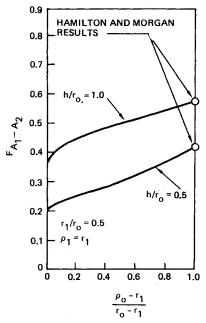


Fig. 5 Shape factor, $F_{\mathrm{d}A_I-A_2}$, from a finite-sized annular disk to the outer wall of a concentric cylindrical enclosure as a function of radial distance parameter $(\rho_0-r_I)/(r_0-r_I)$.

area to a circular disk

$$F_{\text{dA}_I\text{-disk}} = \frac{1}{2} \left[I + \frac{r_o^2 - r^2 - h^2}{\sqrt{(r_o^2 + r^2 + h^2)^2 - 4(r^2 r_o^2)}} \right] \tag{7}$$

is subtracted from one.

Shape factors between a differential area and the outer wall of a concentric cylindrical enclosure are plotted against the quantity $(r-r_1)/(r_o-r_1)$, for several values of r_1/r_o and h/r_o in Fig. 4.

Shape factors between a finite-sized annular disk and the outer wall of a concentric cylindrical enclosure can be determined by integrating Eq. (5) in the following manner:

$$F_{A_I \cdot A_2} = \frac{2}{(\rho_0^2 - \rho_I^2)} \int_{\rho_I}^{\rho_0} r F_{dA_I \cdot A_2} dr$$
 (8)

The integral in this expression can be evaluated numerically using standard techniques such as Simpson's Rule. Typical results are shown in Fig. 5. Note that for $\rho_1 = r_1$ and $\rho_0 = r_o$, the results are identical to those given for configurations A7 and A8 in Ref. 2.

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